IN THE SPECIFICATION:

Please insert the following on Page 1, line 1 of the specification (below the title):

--BACKGROUND OF THE INVENTION

1. Field of the Invention--

Please revise the first paragraph beginning on Page 1, line 4 of the specification to read as follows:

--During the functional operation of a piston engine, there are fed into the shaft of the piston engine transverse forces which result from the forces of the piston and which tend to bend [[said]] the shaft. The shaft and at least two appertaining pivot bearings therefore have to be constructed in a sufficiently robust manner. In spite of a robust construction and mounting of the shaft, bending of the latter occurs because of the elasticity of the material, a fact which leads to inclined positions of [[said]] the shaft in the region of the bearing sections, and that is particularly the case when [[said]] the bearing sections are at an axial distance from one another. In the case of axial piston engines, in particular, this distance is relatively great and is determined by the axial dimensions of a cylinder block and a drive disc.--

Please insert the following on Page 1, line 23 of the specification:

--2. Discussion of the Prior Art--

Please revise the paragraph beginning on Page 1, line 28 through Page 2, line 3 of the specification to read as follows:

--Because of deflection and the resulting inclined position of the relevant bearing section in the region of the pivot bearing, an inclined position of the bearing ring in [[said]] the pivot bearing also automatically occurs, a fact which leads not only to jamming effects in the pivot bearing but also to one-sided loadings with correspondingly high surface pressures (so-called "edge runners"). As a result of this, the bearing surfaces are subjected to higher loads, a fact which leads to higher wear and to a reduction in the working life of the pivot bearings.--

Please insert the following on Page 2, line 12 of the specification:

--SUMMARY OF THE INVENTION--

Please delete the paragraph on Page 2, lines 19-21 in its entirety.

Please revise the paragraph beginning on Page 2, line 23 of the specification to read as follows:

--In the piston engine according to the invention ~~in accordance with claim 1~~, the axial length of the supporting region is shortened to a central region of the bearing section, and radial play is present between the bearing section and the inner bearing ring in the two outer regions next to [[said]] the supporting region.--

Please revise the paragraph beginning on Page 2, line 30 through Page 3, line 2 of the specification to read as follows:

--In [[the]] a configuration according to the invention ~~in accordance with claim 10~~, the axial length of the supporting region is shortened to a central region of the bearing section, [[said]] the bearing section having a greater diameter in its axial central region than in its outer regions.--

Please revise the paragraph beginning on Page 3, line 4 of the specification to read as follows:

--In [[the]] another configuration according to the invention ~~in accordance with claim 15~~, the axial length of the supporting region is shortened to a central region of the inner bearing ring, and [[said]] the bearing ring has a diameter, in the two outer regions next to the supporting region, which is smaller than in the outer regions.--

Please revise the paragraph beginning on Page 3, line 33 through Page 4, line 6 of the specification to read as follows:

--The length of the shortened supporting region may amount to about 1/4 to 1/2, and in particular to about 1/3, of the length of the bearing section or bearing ring. Tests have shown that these dimensional ranges lead, on the one hand, to a sufficiently large supporting region and, on the other hand, to sufficiently large clearances on both sides of [[said]] the supporting region. The supporting region itself may be of cylindrical construction on its superficies.--

Please revise the paragraph beginning on Page 4, line 8 of the specification to read as follows:

--On the bearing section, the outer regions may be constituted by longitudinal regions of the relevant bearing section which are step-shaped or continuously narrowed towards the outside. In comparable manner, the outer regions on the bearing ring may be constituted by widened portions that diverge in a step-shaped manner or towards the ends of [[said]] the bearing ring.--

Please insert the following on Page 4, line 30 of the specification:

--BRIEF DESCRIPTION OF THE DRAWINGS--

Please insert the following on Page 5, line 16 of the specification:

--DETAILED DESCRIPTION OF THE INVENTION--

Please revise the paragraph beginning on Page 5, line 17 through Page 6, line 2 of the specification to read as follows:

--The exemplary piston engine, which is designated, as a whole, by 1, has a housing 2 in whose interior 3 a swash plate 4 and a cylindrical drum 5 are disposed side by side. Disposed in [[said]] the cylindrical drum 5, in a manner distributed uniformly over the periphery, are piston holes 6 which, in the present exemplified embodiment of an axial piston engine, extend substantially parallel to a central axis 7 of the cylindrical drum 5 and are open on that end face 5a of said cylindrical drum 5 which faces towards the swash plate 4. Mounted in a substantially axially displaceable manner in the piston holes 6 are pistons 9 which are preferably cylindrical and which, with their piston heads 9a, delimit working chambers 11 in the cylindrical drum 5 in the direction of the swash plate 4. The foot ends 9b of the pistons 9, which foot ends face towards [[said]] the swash plate 4, are each supported on the latter by a joint 12, under which circumstances sliding blocks 13 may be present, between which blocks and the foot ends 9b are disposed the joints 12, which are preferably constructed as ball joints 12 with a spherical head and a spherical recess.--

Please revise the paragraph beginning on Page 6, line 27 of the specification to read as follows:

--The drive shaft 19, which passes through the cylindrical drum 5 in a bearing bore 23, is rotatably mounted and sealed in bearing recesses in the housing bottom 2b and the cover 2d by means of suitable pivot bearings 25, 26, for example plain bearings or, in particular, rolling bearings, [[said]] the shaft passing axially through the housing bottom 2b and protruding from the latter with a drive pin 19a.--

Please revise the paragraph beginning on Page 7, line 1 of the specification to read as follows:

--In the present exemplified embodiment of the piston engine 1 as a swash-plate engine, the cylindrical drum 5 is disposed in a non-rotatable manner on the drive shaft 19 by means of a rotary-entrainment connection 27, for example a toothed clutch, the [[said]] the drive shaft passing through the swash plate 4, which is disposed fixedly on the housing bottom 2 or constructed therein, in a through-hole 4a. In the present exemplified embodiment, the cylindrical drum 5 rotates, when in functional operation, relative to the swash plate 4, the pistons 9 being displaced longitudinally in the direction of the working chambers 11 and back.--

Please revise the paragraph beginning on Page 8, line 27 of the specification to read as follows:

--If the supporting region 28 is of cylindrical construction, slight compression stresses can occur with the bearing ring 26a can occur in the event of deflection of the drive shafts 19 in the supporting region 28, as a result of which [[said]] the bearing ring 26a can be stretched outwards slightly in its central region, as Figure 4 shows diagrammatically in chain-dotted lines.--

Please revise the paragraph beginning on Page 9, line 1 of the specification to read as follows:

--The exemplified embodiment in accordance with Figure 5, in which parts which are the same or comparable are provided with the same reference symbols, differs from the exemplified embodiment described above through the fact that it is not the bearing section 19c but the bearing ring 26a which has, in its central region a, the supporting region 28a, next to which in the outer regions b, c said bearing ring 26a is widened internally, either in a step-shaped manner or in a manner diverging towards its ends. This results, in each of the outer regions b, c, in radial play, or an annular clearance 29a, 29b, between the cylindrical superficies of the bearing section 19c and the outer regions b, c. In the event of deflection of the drive shaft 19, the end regions of [[said]] the bearing section 19c are able to dip into these clearances 29a, 29b, with the advantages described above.--